

A Granular Flow Model for Tumbling Mills: Dry Mill Power



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Submission: 📅 March 19, 2018; Published: 📅 March 29, 2018

Abstract

We develop a simple granular flow model for the dissipative power density and power draw in dry tumbling mills. The model is based on a Herschel-Buckley like granular rheology with a yield stress proportional to the pressure. Data obtained from Positron-Emission-Particle-Tracking is used to correlate dependence on fill, speed and lifters.

Keyword: Power; Efficiency; Granular; Rheology; PEPT

Introduction

Tumbling mills play a crucial role in many industries. Such mills account for some 60% of operating costs, but are only <5% efficient in comminution [1]. Hence even incremental increases in efficiency hold the promise of enormous savings. To achieve improved efficiency it is natural to seek a better understanding of the processes taking place inside the mill. Unfortunately, the aggressive opaque environment of a milling operation does not afford direct access to the mechanical environment. Consequently, most models are formulated from empirical measurements. Such empirical models can be tuned to provide highly robust descriptions; however, their specialised nature negates the ability to extrapolate beyond the mode of operation for which they were determined.

In recent year's novel techniques like Positron-Emission-Particle-Tracking (PEPT) [2], and X-ray imaging, [3], have allowed researchers an in-situ perspective of particle dynamics deep within the charge body. Both systems are specialised to provide 3D trajectory fields, however, unlike the X-ray imaging system PEPT allows the use of real charge within a scaled industrial mill. PEPT uses particles labelled with β^+ emitters (the tracer), the subsequent annihilation of the positron with an electron produces 511 keV back-to-back gamma pairs. Detection of a few such pairs provides the raw data for tracking the in-situ flow field of the tracer through opaque environments. Through PEPT one can build up time-averaged velocity and solids concentration fields, as illustrated in Figure 1. In particular, the free surface of the charge is quantitatively

discernable as a sharp transition in the solids concentration. With PEPT one can also find the shear rate distributions, but cannot directly access the shear stress. Recently PEPT has been used in the development of a power draw model based on the lever arm approach, Bbosa et al. [4].

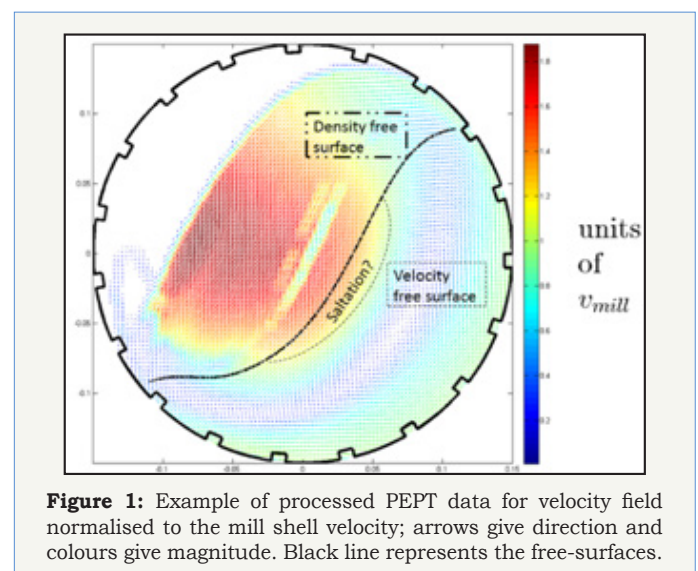


Figure 1: Example of processed PEPT data for velocity field normalised to the mill shell velocity; arrows give direction and colours give magnitude. Black line represents the free-surfaces.

In this paper we use PEPT data in conjunction with a new granular flow model for the charge in the tumbling mill. The use of a continuum description of granular material is hardly new; it underlies the Eulerian-Eulerian approach to multi-phase flows in Computational Fluid Mechanics (CFD). Rajchenbach [5] used the

Bagnold [6] rheology to study avalanches of mill charge. Taberlet et al. [7] assumed a Newtonian-like rheology supplemented with end wall friction to describe the free surface shape. Common to both is the assumption of rigid body motion below the equilibrium surface in contradiction to the definition of the equilibrium surface as the velocity does not vanish there.

What is novel here is, first, that we use a simple granular rheology inspired by recent developments in granular flow theory which amounts to a Herschel-Buckley model but with a yield proportional to the pressure. Second, by cutting the charge into thin slices in which the flow is treated analogously to an incline we obtain simple formula for shear rate and shear stress—hence also forth edissipative power density—that apply out to the mill shell. The importance of the dissipative power density in comminution is that, unlike the power draw, it gives a measure of the distribution of power available for liberation. Moreover, unlike Eulerian CFD, everything can be implemented on an ordinary laptop.

The balance of this paper is organised as follows: in Section 2 we develop our granular rheology¹. In Section 3 we apply this rheology to the mill. Section 4 describes the PEPT experiments, which we analyse using our model in Section 5. Section 6 gives our conclusions.

Granular Rheology

The theoretical study of granular flows can be traced back more than 60 years to the seminal work of Bagnold² [6,8]. Just over a decade ago the appearance of the seminal papers of Midi & da Cruz et al. [9,10], and the “granular rheology” due to Jop et al. [11], revitalised the subject³. The essence of the new rheology is that the dimensionless ratio of the shear stress τ to pressure p as well as the solids concentration ϕ are in turn functions of the so-called inertial number I :

$$\frac{|\tau|}{p} = \mu(I) = \mu_s + \mu_k(I), \quad \phi = \phi(I). \quad (0.1)$$

In that vernacular μ is termed the “friction coefficient”. The inertial number is the ratio of the time scale of rearrangement for spheres of diameter d and mass density ρ_m , $d\sqrt{\rho_m/p}$ and the shearing time scale $1/|\dot{\gamma}|$:

$$I = \frac{d|\dot{\gamma}|}{\sqrt{p/\rho_m}}. \quad (0.2)$$

More recently the Jop-type rheology has been shown to be constrained by the criteria of being well-posed, Barker et al. [15].

Now, the Jop et al. [11] split into static and kinetic frictions has a fluid analogy: is related to a Drucker–Prager yield criterion. Since dimensional arguments play a key role in Midi [9] it useful

to see how the Bagnold [6] rheology thereby arises. If one makes the obvious assumption that a granular flow is non-Newtonian then

$$\tau = \eta\dot{\gamma}, \quad \eta = \eta(|\dot{\gamma}|). \quad (0.3)$$

The ratio η/ρ where $\rho = \phi\rho_m$ has dimension (length)²/(time) and since d ($|\dot{\gamma}|$) is the only intrinsic length (inverse time) $\eta/\rho \propto d^2|\dot{\gamma}|$; the dimensionless proportionality constant can only be a function of ϕ whence $\eta \sim \rho_m d^2 k(\phi)|\dot{\gamma}|$. According to Bagnold, 1956 $k(\phi) = 0.013\lambda^2(\phi)$ where $\lambda(\phi) \equiv 1/[(\phi/\phi^*)^k - 1]$ is the “linear grain concentration” and $\phi^* = 0.65$ (close to $\phi_0 = 0.634$ for random close packing)⁴.

Of course that isn’t the whole story because it does not include the existence of the yield stress. To fix that one most simply imitates the $n = 2$ Herschel-Buckley generalization of the Bingham model:

$$\eta = \begin{cases} \infty, & |\tau| < \tau_0 \\ \frac{\tau_0}{|\dot{\gamma}|} + K|\dot{\gamma}|, & |\tau| > \tau_0 \end{cases}. \quad (0.4)$$

For the familiar cases of Bingham fluids, the yield stress τ_0 is an independent constant determined by the cohesive interactions of the constituents. Clearly that is not the case for e.g. dry smooth spherical beads, and one reasons that τ_0 should vanish in the absence of compression in that case. The simplest way to model that is to take the yield stress proportional to the pressure: $\tau_0 = p\mu_c$.

One thus arrives at

$$\eta = \begin{cases} \infty, & |\tau| < p\mu_c \\ \frac{p\mu_c}{|\dot{\gamma}|} + \rho D^2 |\dot{\gamma}|, & |\tau| > p\mu_c \end{cases}. \quad (0.5)$$

Here we have introduced a rheological diameter by $D \equiv d\sqrt{k(\phi)/\phi}$ such that $\rho = \rho_m\phi$ is the superficial mass density. In the context of the Bagnold [8] bed load theory the yield term can be viewed as a shear thinning fluid $\eta_f = p\mu_c/|\dot{\gamma}|$

Mill Model

Consider now the application of our rheology to the tumbling mill. We construct at a point a local coordinate system with tangent to the free surface⁵, subtending angle θ with \hat{x} , and \hat{x}' pointing into the bed, as illustrated in Figure 2.

The lab coordinates are related to the local ones as

$$x = x_f + x' \sin \theta, \quad y = y_f - x' \cos \theta \quad (0.6)$$

On the free surface

$$\tan \theta = \frac{dy_f}{dx_f} \quad (0.7)$$

¹For a review of granular rheology in mills see Govender [12].

²For a retrospective see Hunt et al. [13].

³For a review see Forterre & Pouliquen [14].

⁴We note that an alternative to Pouliquen et al. [16] is $I = 3\phi_s/[\Delta\phi\lambda(\phi)]$ where $\Delta\phi = 0.111$ the difference of random close and simple cubic is packing.

⁵Close inspection of Figure 1 reveals that there is a “velocity free surface” under the “density free surface”, separated by a thin layer—we ignore this nicety.

⁶It should be said that these assumptions are reasonable approximations in the central region—see Figure 1a, but must break down near the shoulder and toe as those regions involve recirculation of the charge.

Hence, as y_f is a function of x_f , the independent variables are x_f, x' . Now we will make the small curvature or lubrication approximation: with $x' = h$ the depth that the mill shells, s_f the displacement along the free surface, and $|R_f|$ the radius of curvature of the free surface

$$(dx')^2 + (dy')^2 = (ds_f)^2 + (dx')^2, x'/|R_f| \leq h/|R_f| \ll 1 \quad (0.8)$$

Assuming that the flow is stationary, parallel to the free surface and independent of s_f or z , the equations describing the flow are

$$\begin{aligned} \dot{\gamma} &= \frac{dv}{dx'} \\ 0 &= \frac{\partial}{\partial x'} \tau - \rho g \sin \theta \\ 0 &= -\frac{\partial p}{\partial x'} + \frac{\partial}{\partial s_f} \tau + \rho g \cos \theta \end{aligned} \quad (0.9)$$

Assuming ϕ also constant, integrating into the bed with free-surface boundary conditions and the rheology (0.5) gives

$$\begin{aligned} \tau &= \left(p\mu_c + \rho D^2 \left[\frac{dv}{dx'} \right]^2 \right) = \rho g \sin \theta x' \quad (0.10) \\ p &= \rho g \cos \theta x' \{ + \text{order } x'/R_f \} \end{aligned}$$

Hence the shear rate is

$$\dot{\gamma} = \frac{dv}{dx'} = \sqrt{\frac{g}{D^2} (\sin \theta - \mu_c \cos \theta)} x' \quad (0.11)$$

We note that the friction and inertial number are depth independent:

$$I = \frac{d}{D} \sqrt{\frac{\tan \theta - \mu_c}{\phi}}, \mu = \tan \theta = \mu_c + \left[\frac{ID}{d} \right]^2 \phi \Leftrightarrow k(\phi) = \left[\frac{D}{d} \right]^2 \phi \quad (0.12)$$

Since our local coordinates are a translation and rotation, the dissipative power density is

$$P = \tau \dot{\gamma} = \rho g \sin \theta \sqrt{\frac{g}{D^2} (\sin \theta - \mu_c \cos \theta)} x'^3 \quad (0.13)$$

Integrating (0.11) subject $v(h_e) = 0$ to at the equilibrium surface

$$v(x') = \frac{2}{3} \sqrt{\frac{g}{D^2} (\sin \theta - \mu_c \cos \theta)} \left[x'^{3/2} - h_e^{3/2} \right] \quad (0.14)$$

Continuing this description out to the mill shell, we have that the flux per unit mill length vanishes:

$$0 = \int_0^h v_y dx = \frac{2}{3} \sqrt{\frac{g}{D^2} (\sin \theta - \mu_c \cos \theta)} \left[\frac{2}{5} h^{5/2} - h h_e^{3/2} \right] \quad (0.15)$$

Then we have $h_e = (2/5)^{2/3} h = 0.543h$. In general, the x' axis is not aligned with the normal to the mill shell; see Figure 2. Further, even if it is there may be slip—see Figure 1a. Denote $\omega_s = \omega \Omega$ as the angular 'slip speed'. Hence the velocity $x' = h$ at is related to the mill radius R and mill angular speed ω by

$$R\omega_s \cos \alpha = \frac{2}{5} \sqrt{\frac{g}{D^2} (\sin \theta - \mu_c \cos \theta)} h^{3/2}; \quad (0.16)$$

The angle α is related to x_f, y_f as

$$x_f^2 + y_f^2 = R^2 + h^2 - 2Rh \cos \alpha \quad (0.17)$$

Finally, the dissipative power density can be usefully re-expressed as

$$P = \frac{5}{2} R \omega_s \rho g \sin \theta \cos \alpha \left(\frac{x'}{h} \right)^{3/2} \quad (0.18)$$

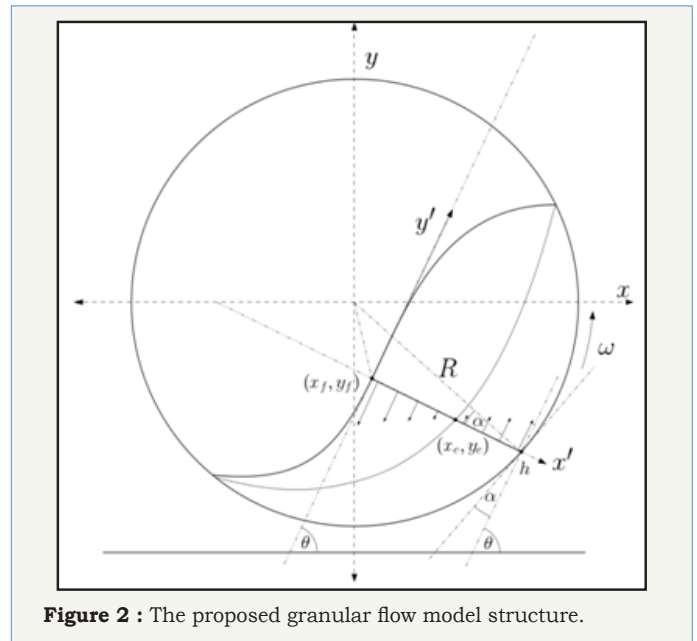


Figure 2 : The proposed granular flow model structure.

Note that this is independent of particle size. Because of the non-Newtonian rheology this grows faster than the shear rate that is often used as guide to liberation in comminution. To obtain the total power per unit length one needs to observe the global measures are different owing to the boundaries

$$\begin{aligned} \frac{dP}{dz} &= \int_{x_{fT}}^{x_{fS}} \frac{dx_f}{\cos \theta} \int_0^h P = \int_{x_{fT}}^{x_{fS}} dx_f \tan \theta \omega_s \rho g R h \cos \alpha \\ &= \omega_s \rho g \int_{x_{fT}}^{x_{fS}} dx_f \left(\frac{dy_f}{dx_f} \right) \left(\frac{R^2 + h^2 - r_f^2}{2} \right) \end{aligned} \quad (0.19)$$

Integrating by parts

$$\frac{dP}{dz} = R \omega_s \rho g h_{CoC} (y_{fS} - y_{fT}) \quad (0.20)$$

Herein h_{CoC} is the depth measured along a radial line through the Centre of Circulation, and $\tan \theta_{CoC}$ is the free surface slope there, while y_{fS} (y_{fT}) is the mill shell y -coordinate at the shoulder (toe). When the mill is static the charge forms a segment subtending $\alpha_T + \alpha_S$; the stationary flow displaces the shoulder and toe such that $y_{fS} = R \sin(\theta_{CoC} - \alpha_S)$ and $y_{fT} = -R \sin(\theta_{CoC} + \alpha_T)$ where $\cos \alpha_{ST} = h_{CoC}/2R$. Hence

$$\frac{dP}{dz} = R \omega_s \rho g h_{CoC}^2 \sin \theta_{CoC} \quad (0.21)$$

Since what goes in must come out, (0.21) is also a formula for power draw per unit length.

The PEPT Experiments

The experiments were performed at the PEPT facility, iThemba Labs Cape Town, with the Siemens EXACT3D scanner. The experimental rig, Figure 3, consisted of a mill constructed from

High Density Polyethylene (HDPE), a DC drive with step-down gear box and torque sensor. The mill shell and lifters were manufactured from HDPE (specific density of 0.95) while the lifters were made from aluminium (Figure 3).

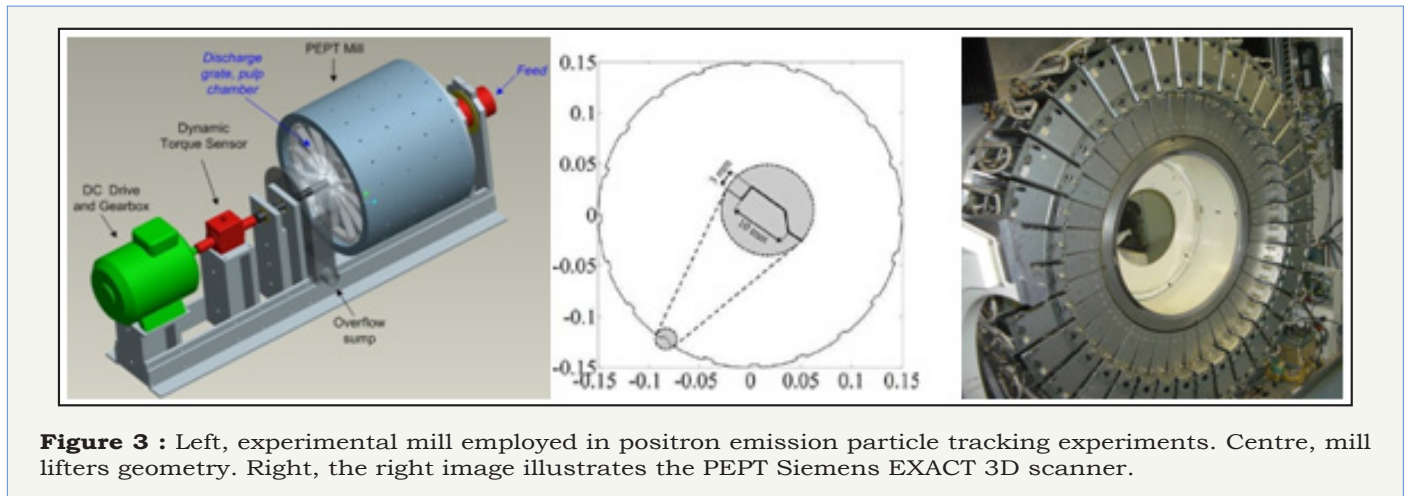


Figure 3 : Left, experimental mill employed in positron emission particle tracking experiments. Centre, mill lifters geometry. Right, the right image illustrates the PEPT Siemens EXACT 3D scanner.

We will consider PEPT data with lifters was acquired by scanning a model mill of $R=0.150$ m and length of $L=0.3$ m with lifters varying in height (1.5, 3, 6 and 10 mm), containing 20%, 30% and 40% volumetric filling of 5 mm borosilicate glass balltini and introducing a drilled and activated representative glass bead tracer to track with the Siemens EXACT3D scanner. The mill was run at 55, 70 and 85 % of critical speed. The tracer was activated with a resin bead that was subjected to an ion-exchange reaction using ^{68}Ga isotope with an activity of no more than 1mCu, allowing for a tracking time of up to 2 hours.

The charge velocity field can be directly obtained by differentiating the trajectory data. The determination of the solids concentration distribution is based on the methodology of Wildman et al. [16]: the mill volume is partitioned into rectangular voxels; the number density in a voxel j of the mill is given by

$$n(j) = \frac{NF(j)}{V_j} \quad (0.22)$$

Here N is the number of particles, $F(j)$ is the residence time fraction of the tracer in voxel j , and V_j is the volume of the voxel. With t_j is the time spent by the particle in a given voxel and T the total system observation time, the residence time fraction - or the fraction of total observation time a particle spends in a particular segment - is given by:

$$F(j) = \frac{t_j}{T} \quad (0.23)$$

The solids concentration ϕ in voxel j is then given by:

$$\phi(j) = n(j) \frac{\pi d^3}{6} \quad (0.24)$$

Model PEPT Data Analysis

To begin, it is useful to take the mill angular speed ω in units of the critical speed $\omega_c = \sqrt{g/R}$ for centrifuging, and lengths in units of the mill radius R . These natural units amount to setting $R=g=1$ in the formula (Figure 4).

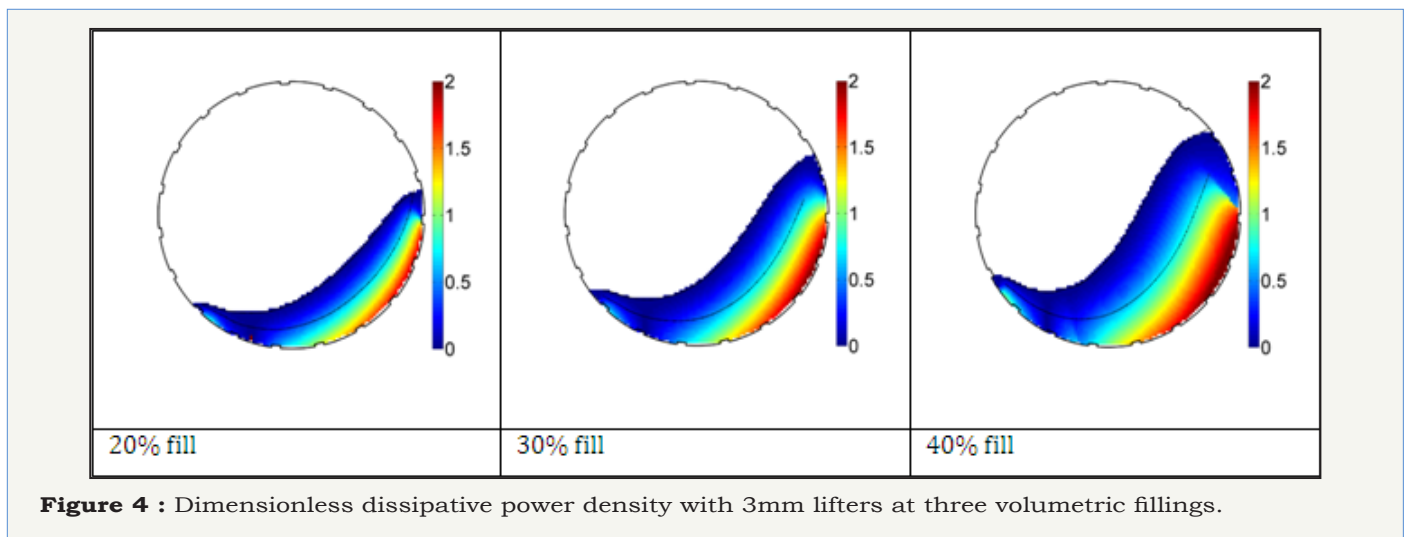


Figure 4 : Dimensionless dissipative power density with 3mm lifters at three volumetric fillings.

We start with the dissipative power density (0.18) since this is independent of rheological size. Indeed, the dimensionless

$$P/(R\omega_s\rho g) = (5/2)\sin\theta\cos\alpha(x'/h)^{3/2}$$

with α taken from PEPT can only depend on volumetric fill. The results are displayed in Figure 4. One notes similarities to the power draw density maps of Bbosa et al. [4]—the primary difference is that the power draw maps show an enhancement around the CoC that is absent in the dissipation maps. Since it involves lifters and the speed typical in comminution it is most telling that this is strongly peaked in a short narrow band near the mill shell. One thus

begins to understand why the comminution tumbling mill has a low efficiency in liberation.

In application, one needs to know not only the mill radius and charge density, but also the dependence of slip on fill and lifters. We have measured the slip just above the lifters along the radial line through the centre of circulation (Coc). Further, one needs correlating of h_{Coc} and $\cos\theta_{Coc}$. The results are summarised in Figure 5, along with the predicted dimensionless dissipative power per unit length (Figure 5).

$$(dP/dz)/(R\omega\rho g) = \Omega h_{Coc}^2 \sin\theta_{Coc}$$

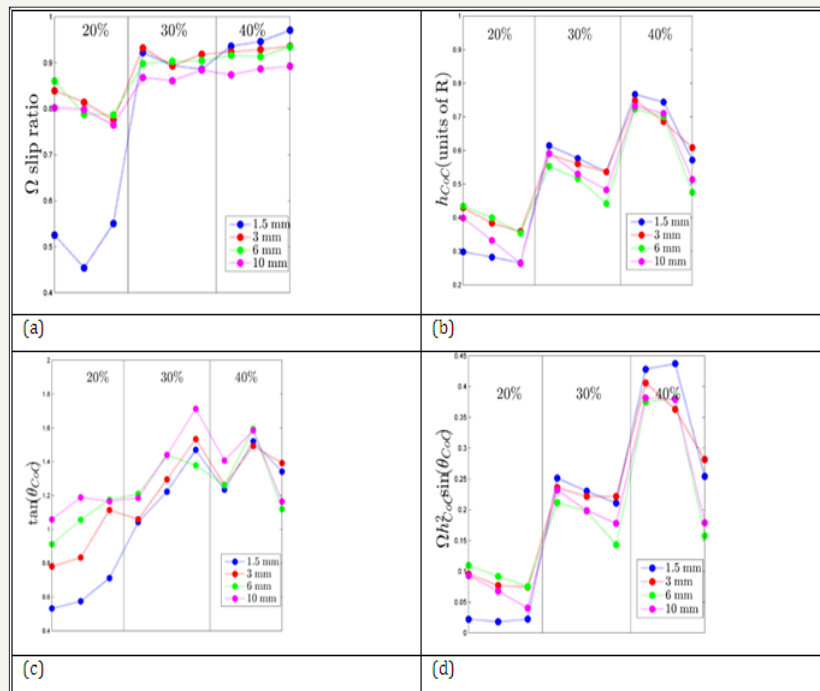


Figure 5: Measured (a) slip Ω , (b) bed depth h_{Coc} (c) free surface slope $\tan\theta_{Coc}$ and (d) dimensionless power per unit length for different fills and lifter sizes.

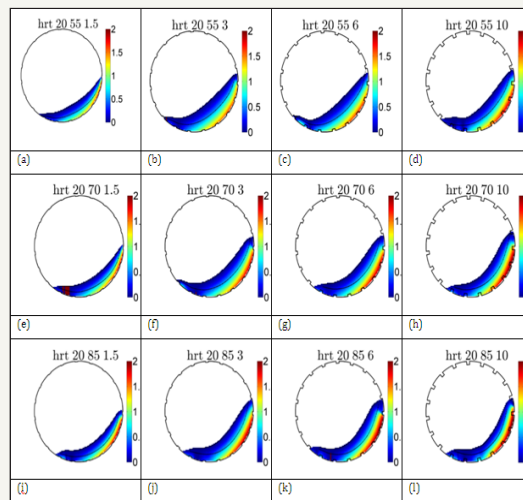


Figure 6 : Dimensionless dissipative power density $P/(R\omega_s\rho g)$ for 20% fill with 1.5mm, 3mm, 6mm, and 10mm lifters (left to right) at 55%, 70% and 85% critical speed (top to bottom).

The trend in Figure 5 is clear: reducing the fill significantly reduces the total dissipative power and, since what goes in must come out, also the power draw. At 20% fill, the 1.5mm lifters for 5mm beads are obvious outliers; to confirm this we display the full suite of maps in Figure 6—one notes the higher liberation zone is squeezed into a vanishingly thin strip. In the other cases, higher lifters and/or speed enhance the higher liberation zone.

Conclusions

In this paper we have developed a granular flow model for the tumbling mill, based upon a $n = 2$ Herschel-Buckley type rheology with a yield stress proportional to the pressure. Using a local coordinate system tangential to the free surface we formulated and solved the non-Newtonian Navier-Stokes equations to obtain shear rate and shear stress—hence also the dissipative power density. We further obtained an expression for the total dissipative power—which is to say power draw-per unit length. We have not validated the total power here due to absence of corroborating measurements for the extensive data set considered.


While a number of simplifying assumptions and approximations were involved, the model's basic equations can be quickly solved by anyone on any laptop (and some tablets) provided a model for the free surface such as Taberlet [7] is assumed. Finally it must be reminded that the model is restricted to dry systems in the sense that the Bagnold [6] number is large; the extension to wet systems involves a number of issues beyond those addressed here. Similarly, understanding the free surface necessitates a more refined treatment of the structures exposed in Figure 1, similar to Bagnold's [8] bed load theory.

Acknowledgement

This work was supported by Anglo American. The work of G. B. Tupper was partially supported by a grant from the National Research Foundation.

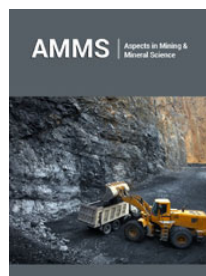
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